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Predicting Tractor Diesel Fuel Consumption

Authored by Robert "Bobby" Grisso, Extension Engineer, Biological Systems Engineering, Virginia Tech

Introduction

Ability to predict tractor fuel consumption is very useful for budgeting and management. The objective of this factsheet is to develop relationships using field measurements and Nebraska Tractor Test Laboratory results to estimate tractor fuel consumption. Using these equations, farmers can estimate and compare the fuel consumption for different operating and loading conditions.

Depending on the type of fuel and the amount of time a tractor or machine is used, fuel and lubricant costs will usually represent at least 16 percent to over 45 percent of the total machine costs. Thus, fuel consumption plays a significant role in the selection and management of tractors and equipment used in agriculture. Currently, most budget models use a simplified methods for estimating fuel consumption. Better estimates representing actual field operations are needed to compare machinery management strategies.

The worth of a tractor is assessed based on work output and the cost associated with completing the task. Drawbar power is defined by pull (or draft) and travel speed. An ideal tractor would convert all fuel energy into useful work at the drawbar. However, due to power losses, not all fuel energy is converted into useful work.

Efficient operation of farm tractors may depend on: (1) maximizing the fuel efficiency of the engine and the mechanical efficiency of the drivetrain, (2) maximizing tractive efficiency of the traction devices, and (3) selecting an optimum travel speed for a given tractor-implement system. This factsheet focuses on methods to estimate and improve fuel efficiency of a diesel power unit.

The Nebraska Tractor Test Laboratory (NTTL) has a long history of testing tractors and disseminating power and fuel consumption data. The NTTL is the official testing station for agricultural tractors in the United States. Tractors manufactured in the United States and other countries are tested, and NTTL publishes the test results. During standardized tests, the power is calculated and the corresponding fuel consumption is measured. The power at the power takeoff (PTO) is calculated from the torque and speed at the PTO. Drawbar power is calculated from the drawbar pull and the forward speed of the tractor. For more details and for a sample test report, see *Using Tractor Test Data for Selecting Farm Tractors*.

Terminology

Tractor manufacturers specify power output at several tractor locations, such as power takeoff, drawbar, hydraulic outlets, and electrical outlets. For each tractor model, the rated power output is measured at the rated engine speed. Typically, this power is measured at the PTO, and in this factsheet, it is referred to as "rated PTO power." For most modern tractors, the rated power will not be the maximum available power. Most modern engines often produce more power because they can be operated at speeds other than rated speeds.

Fuel consumption is measured by the amount of fuel used during a specific time period. The most common measure of the energy efficiency of a tractor is referred to here as "specific volumetric fuel consumption" (SVFC), which is given in units of gallons per horsepower-hour (gal/hp-h). Specific volumetric fuel consumption is generally not affected by engine size, and it is used to compare the energy efficiencies of tractors with different size engines and under different operating conditions. SVFC for diesel engines typically ranges from 0.0476 to 0.1110 gal/hp-h.

For ease of computation, the reciprocal of SVFC is often used and is referred to here as "specific volumetric fuel efficiency" (SVFE) with units of horsepower-hours per gallon (hp-h/gal), with corresponding ranges from 12 to 21 hp-h/gal.

The NTTL reports the SVFE for several drawbar load tests, rated PTO speed, and varying PTO power tests. Table 1 shows sample data from an NTTL report. The SVFE for this test is shown under the columns labeled with units of hp-h/gal (kW-h/L). For example, at rated engine speed, the tractor shown in Table 1 developed 115.96 PTO horsepower with an SVFE of 17 hp-h/gal.

The data from NTTL Report 1725 (shown in Table 1) is used to demonstrate the difference of load level. For the drawbar performance at "75% of Pull at Maximum Power," the engine speed was 2,190 rpm, and the SVFE was 12.80 hp-h/gal. The corresponding test with a reduced throttle setting had an engine speed of 1,665 rpm and an SVFE of 14.63 hp-h/gal. The SVFC was calculated as 0.0781 gal/ hp-h for full throttle and 0.0684 gal/hp-h for the reduced throttle test. The decrease in SVFC was 12.4 percent, while the engine speed was reduced by 24 percent. Similarly, the "50% of Pull at Maximum Power" tests resulted in a 24 percent reduction in engine speed and a 15.8 percent decrease in SVFC.

Tractor Fuel Efficiency Improvements

Newer tractors are generally more efficient than previous models. Improvements in fuel efficiency during the last 20 years prove this point. Figure 1 shows the average and maximum specific fuel consumption of tractors tested from 1980 to 2000. Models tested in 2000 averaged 16.5 hp-h/gal, compared to an average of 14.5 hp-h/gal for models tested in 1980. The fuel savings of 10 to 15 percent became possible because

of improved engine/transmission design and improved ability to match tractors and implements for given field conditions. Today's tractors have more electronic controls for more efficient delivery of power to the PTO, drawbar (for pulling), and to the hydraulic system.

The reduction in fuel efficiency seen during years 1991, 1998, and 2004 (figure 1) is attributed to the new emission requirements that went into effect for off-road vehicles. Even though Environmental Protection Agency (EPA) regulations initially challenged engine designers, fuel efficiency has improved significantly since then. All tractors are not equal in fuel consumption. The maximum value line in figure 1 represents the fuel efficiency for the most fuel-efficient tractor for that year. The fuel consumption data is an important consideration during the selection and purchase of a tractor.

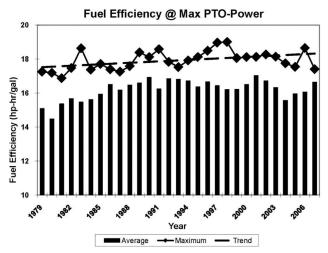


Figure 1. The average and maximum fuel efficiency (hp-hr/gal) for almost 30 years. This chart shows results averaged from annual NTTL reports (typically 10-30 tractors per year).

Fuel Consumption Estimates

ASABE Standards are widely used for estimating fuel consumption for determining cost of operations. The most widely used relationship for estimating fuel consumption in gallons per hour (gal/h) is

$$Q_{AVG} = a' \cdot P_{PTO} \tag{1}$$

where

 Q_{AVG} = average diesel consumption (gal/h), P_{PTO} = rated PTO power (hp), a' = 0.044 gal/hp-h.

Bowers (2001, e-mail correspondence) stated that equation (1) was developed based on PTO power test results from the Nebraska Tractor Test reports during the mid-1970s. The fuel consumption (gal/h) over the varying PTO power tests (approximately 100 percent, 85 percent, 65 percent, 45 percent, 20 percent, and 0 percent of rated PTO power) were averaged, then the average was divided by the rated PTO power. For this reason, the annual fuel consumption estimates using this method give fuel consumption based on the assumption that the tractor is operated under the same load pattern for equal time. Due to this assumption, this method typically underestimates fuel consumption.

Field Measurement of Fuel Consumption

Farmers may consider numerous ways to estimate and reduce fuel consumption. The first step is to determine how much fuel is being used for a particular field operation and compare it to average usage. This measurement can be completed by filling the fuel tank of the tractor before and after a field operation, noting the number of acres covered. The number of gallons used, divided by the number of acres covered, gives the fuel consumption in gallons per acre (gal/ac). The resulting estimate of fuel consumption can be compared to the values listed in table 2¹. If measured fuel consumption is higher than the estimated average, the following tips should be considered to reduce fuel consumption:

- Eliminate one or more tillage/field operations.
- Substitute one type of tillage operation for another (e.g., chisel for plow).
- Combine tillage and other operations into one pass over the field.
- Optimize tractor performance (proper amount and distribution of ballast, tire inflation, etc.).
- Use "gear-up, throttle-down" (GUTD) practices.
- Properly match tractor and implement.
- Optimize field efficiency with controlled traffic and navigation aids.
- Ensure sufficient machine maintenance.

Better Fuel Consumption Estimates

Two equations ("general" and "specific") have been developed for estimating fuel consumption from

Nebraska Tractor Test results. Both are based the relationships between engine load and engine speed (rpm) developed from the tractor test results. Results from more than 500 tractors were used to develop the "general" equations. The "specific" equations used the other method derived from the test results of test conducted on one specific tractor.

Fuel Consumption Estimates for "General" Tractors

Fuel consumption is estimated from reduced engine load and from full-throttle data (along the governor response power curve). *ASABE Standards* general fuel consumption equations for compression ignition engines were developed as follows:

$$Q_F = (0.0434X + 0.019) \cdot P_{PTO}$$
(2)

where

- Q_F = diesel fuel consumption at partial load and full throttle (gal/h),
- X = fraction of equivalent PTO power available (decimal).

$$X = P/P_{RATED}$$

where

P = equivalent PTO power required by current operation (hp),

 P_{PTO} = rated PTO power available (hp).

$$Q_R = Q_F \cdot [1 - (N - 1) \cdot (0.45X - 0.877)]$$
(3)

where

- Q_R = diesel fuel consumption at partial load and reduced throttle (gal/h),
- N = ratio of reduced- and full-throttle engine speeds at operating load (decimal).

$$N = RPM_{PT} / RPM_{FT}$$

where

 RPM_{PT} = engine speed at partial throttle at reduced engine speed (rpm),

 RPM_{FT} = full-throttle engine speed (rpm).

This "general" equation is useful for predicting fuel consumption by diesel engines under different load and speed conditions.

¹Table 2 reports fuel consumption values for many field operations summarized from various states. Although data is not included from every state and is dated, the values may be useful as guidelines to determine whether fuel usage for a particular operation is reasonable.

Fuel Consumption Estimates for "Specific" Tractor Models

The second equation for estimating fuel consumption was developed from the general model (equations (2) and (3)) and was used to derive the coefficients for specific tractor models. The method uses the NTTL test results, and the defining locations of the fuel consumption parameters are shown in figure 3. The equation for a specific tractor is:

$$Q = (aX + b) \cdot [1 - (N - 1) \cdot (cX - d)] \cdot P_{_{PTO}} \quad (4)$$

where

a, b, c, and d coefficients are defined in appendix A,

Q = diesel fuel consumption (gal/h),

- X = ratio of equivalent PTO power to rated PTO power (decimal),
- N = ratio of reduced- and full-throttle engine speeds at operating load (decimal),

 P_{PTO} = rated *PTO* power (hp).

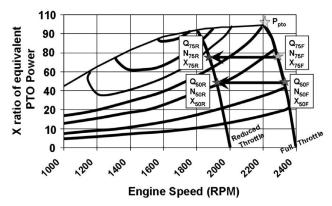


Figure 3. Engine map with the tractor test information and notations described.

Coefficients (a, b, c, and d) were computed for specific tractors tested during the period 1986-2008. More than 770 tractors with complete drawbar tests (both 50 percent and 75 percent pull tests) were considered. The use of coefficients specifically calculated to a specific tractor greatly improved the accuracy of prediction. The computation of coefficients for a specific tractor (of your choice) as shown in appendix A.

Fuel Consumption Comparison

Estimated fuel consumption for the John Deere 7610 (NTTL 1725) is compared against measured fuel consumption during the Nebraska tractor test in table 3. The fuel consumption – estimated using equation (4) – agreed well with measured fuel consumption. On the other hand, estimates using the "general" model were found to be slightly higher. Estimates from the traditional method were consistently lower than the actual fuel consumption for full-load tests. The best estimates were from the "general" method. Both showed the benefits of GUTD practices.

"Specific" Model Application

The NTTL report can be a very useful resource for computing the coefficients in equation (4). As shown in the example in appendix A, the coefficients can be derived directly from the NTTL reports. To use the "specific" model to predict fuel consumption for a given field operation, ensure that the current engine and the engine reported in the NTTL report are adjusted in the same manner. The high-idle (full-throttle, no-load) engine speed shown in the "Varying Power and Fuel Consumption" section of the NTTL report must be compared with

	Fuel consumption (gal/h)				
	Traditional estimate (eq (1))	General prediction (eq (2)-(3))	Specific prediction (eq (4))	Actual test data	
PTO power test, rated	5.102	7.236	6.882	6.821	
Drawbar test, maximum pull, rated	5.102	7.236	6.882	6.714	
Drawbar test, 75% pull, full throttle	5.102	6.162	5.916	5.916	
Drawbar test, 75% pull, reduced throttle	5.102	5.397	5.183	5.185	
Drawbar test, 50% pull, full throttle	5.102	4.895	4.776	4.776	
Drawbar test, 50% pull, reduced throttle	5.102	4.149	4.027	4.031	

*Fuel consumption measured and predicted by the "traditional," "general," and "specific" models for Nebraska OECD Tractor Test 1725 – Summary 225 (shown in Table 1).

the current engine setting. For example, the high-idle engine speed shown in the NTTL report example (Table 1) is 2,267 rpm. The user's high-idle engine speed (RPM_{HI}) can be determined by observing the engine speed at full throttle and no load. If the high-idle engine speed is significantly different from the NTTL reported speed, the difference between actual and predicted fuel consumption may be significant.

To determine the engine speed during the field operation (RPM_{FES}) , the operator must record the engine speed with the throttle fully open and under typical field load. This engine speed is used to estimate the ratio of rated power being used. The rated engine speed (RPM_{RATED}) is typically found on the tractor's tachometer, operator's manual, or NTTL report.

Using the relationship below, the estimated ratio of rated power that is being used during field operation is computed:

$$X = (RPM_{HI} - RPM_{FES})/(RPM_{HI} - RPM_{RATED}) \quad (5)$$

where

 RPM_{HI} = high-idle engine speed (rpm),

 RPM_{RATED} = rated engine speed for the tractor being considered (rpm),

 RPM_{FES} = field engine speed during field operations at full throttle (rpm).

This relationship assumes that the governor response (engine droop on the right side of figure 2) is linear. If this is not the case, use speed ranges found in the "Varying Power and Fuel Consumption" section of the NTTL report to estimate the ratio of rated power.

For example, suppose that a JD7610 model tractor is being used for primary tillage and the field engine speed is 2,150 rpm. From the test report (Table 1) and field testing, the high-idle speed is 2,267 rpm and the rated engine speed is 2,100 rpm. The equivalent power, calculated using equation (5), being used for this operation is 0.7, and using the specific coefficients from the example in appendix A, the fuel-use equation becomes

 $Q = (0.039X + 0.0203) \cdot [1 - (N - 1) \cdot (0.5247X - 0.9345)] \cdot 115.96 \text{ hp}$

At full throttle (N = 1), the estimated fuel consumption for primary tillage with the JD7610 is

 $Q = (0.039 \cdot 0.7 + 0.0203) \cdot 115.96 = 5.52 \text{ gal/h}$

To estimate the savings from employing the gear-up and throttle-down practice (GUTD), the operator can look at several engine reduction possibilities and match with the most appropriate. If the tractor engine speed is reduced by 10 percent and 20 percent (N = 0.9 and 0.8, respectively), the decrease of fuel consumption is estimated as:

$$D_{10} = 1 - \{(0.9 - 1) \cdot [(0.5247 \cdot 0.7) - 0.9345]\}$$

 $D_{10} = 0.943$, or a 5.7 percent fuel savings for reducing the throttle by 10 percent.

Likewise,

 $D_{20} = 0.887$, or an 11 percent fuel savings for reducing the throttle by 20 percent.

In terms of fuel consumption rate,

 $Q_{10} = 5.52 \cdot 0.943 = 5.21$ gal/h, or a savings of 0.31 gal/h,

 $Q_{20} = 5.52 \cdot 0.887 = 4.90$ gal/h, or a savings of 0.60 gal/h.

The easiest way to reduce the engine speed is from the no-load condition (at high idle). A 10 percent and 20 percent engine-speed reduction would require moving the throttle from high idle (2,267 rpm) to 2,040 rpm and 1,813 rpm, respectively.

Estimating Fuel Consumption From Field Operations

The impact of individual operations of cropping systems for a 350-acre grain/pasture cropping system is examined and compared. The farm is separated into 125 acres for haying operations and 225 acres for traditional corn production using a 100 hp tractor. Table 4 provides the operations that the tractor performs for the grain/pasture operations. Note that the tractor is used for slightly more than 240 hours annually. The estimated fuel consumption using the "traditional" method would be 4.4 gal/h, irrespective of the type of operation; the total fuel use on the 350 acres would be 1,064 gallons.

When the "general" model (equations (2) and (3)) is

	Acres								
	per			Full	Reduced	d Gallons	Total		Labor
	hour	\mathbf{X}^{\dagger}	Acres	(gal/h)	(gal/h)	per acre	gal	Save	hours
225-acre, minimum tillage co	orn for g	rain							
15-ft no-till drill (cover crop)	7.6	0.75	225	5.2	4.6	0.68	154	17.8	29.6
30-ft sprayer (2-X)	15.0	0.35	550	3.4	2.9	0.23	125	18.3	36.7
15-ft, 6-row planter	7.0	0.60	225	4.5	4.0	0.64	145	16.1	32.1
125-acre pasture (haying: two	o cuttings	.)							
12-ft mower/conditioner*	6.0	0.50	250	4.1	4.1	0.68	171	0.0	41.7
12-ft tedder	7.1	0.20	250	2.8	2.3	0.39	99	17.6	35.2
12-ft side-delivery rake	7.1	0.20	250	2.8	2.3	0.39	99	17.6	35.2
Round baler*	8.0	0.40	250	3.6	3.6	0.45	113	0.0	31.3
						Total	904	87.4	241.8

Table 4. Fuel consumption estimates for field operations using a 100-hp tractor

*Denotes that because PTO driven, the engine cannot be reduced in speed.

[†]Fraction of equivalent PTO power available.

applied for each operation and the load factor is taken into consideration for different field operations that do not require full power, the fuel consumption is estimated at a total of 904 gallons. It is interesting to note that the gallons per acre estimates can be compared with table 2, and the planting operation appears to consume more fuel than other uses (0.64 versus 0.51 gal/ ac). This approach gives producers potential areas to adjust and manage their systems, as well as to optimize their machinery systems based on operating averages.

When the GUTD practice is used in these operations, with only a 20 percent reduction in speed (some operations can probably be dropped further based on the load level), it is estimated that a fuel savings of 87 gallons is possible. This translates into a fuel savings of 10 percent for this tractor annually.

Conclusion

Three methods for estimating tractor fuel consumption are presented. Comparing estimated and measured fuel consumption shows that the fuel consumption estimated using the "specific" model agrees best with the measured fuel consumption. This model has the capability to predict fuel consumption when the tractor is fully or partially loaded and when the engine speeds are reduced from full throttle.

References

- ASABE Standards: Agricultural Machinery Management Data. ASABE EP496.3. St. Joseph, Mich.: ASABE.
- ASABE Standards: Agricultural Machinery Management Data. ASABE D497.7. St. Joseph, Mich.: ASABE.
- Siemens, J. C., and W. W. Bowers. 1999. Machinery Management: How to Select Machinery to Fit the Real Needs of Farm Managers. Farm Business Management Series. Davenport, Iowa: John Deere Publishing.

Original Sources of Fuel Predictions

Grisso, R. D., M. F. Kocher, and D. H. Vaughan. 2004. Predicting tractor fuel consumption. *Applied Engineering in Agriculture* 20(5): 553-61.

Grisso, R. D., D. Vaughan, and G. Roberson. 2008. Fuel prediction for specific tractor models. *Applied Engineering in Agriculture* 24(4): 423-28.

Additional Reading Materials

Five Strategies for Extending Machinery Life. VCE publication 442-451.

Gear Up and Throttle Down: Saving Fuel. VCE publication 442-450.

Using Tractor Test Data for Selecting Farm Tractors. VCE publication 442-072.

For tractor test information, contact:

University of Nebraska Tractor Test Laboratory P. O. Box 830832 134 Splinter Labs UNL, East Campus Lincoln, NE 68583-0832

(402) 472-2442 Fax: (402) 472-8367 http://tractortestlab.unl.edu/ E-mail: tractortestlab@unl.edu

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 Table 1. Example of a Nebraska Tractor Test Laboratory (NTTL) report.

 The sections show: top, the PTO performance tests; center, the varying power tests; and bottom, the drawbar performance test results. This is taken from the report of Nebraska OECD Tractor Test 1725 – Summary 225 for John Deere 7610 PowerShift.

		Power Take	-Off Performan	се	
Power HP	Crankshaft	Gal/br	lb/hp.hr	Hp.hr/gal	Mean Atmospheric
(kW)	speed rpm	(l/h)	(kg/kW.h)	(kW.h/l)	Conditions
	N	Maximum Powe	r and Fuel Consum	nption	
	Rat	ed Engine Spee	d (PTO speed — 10	006 rpm)	
115.96	2100	6.82	0.413	17.00	
(86.47)		(25.82)	(0.251)	(3.35)	
		Maximun	n Power (2 hours)		
117.27	1801	6.51	0.390	18.00	
(87.45)		(24.66)	(0.237)	(3.55)	
		Varying Power	and Fuel Consum	otion	
115.96	2100	6.82	0.413	17.00	
(86.47)		(25.82)	(0.251)	(3.35)	Air Temperature
102.08	2170	6.33	0.435	16.13	75°F
(76.12)		(23.95)	(0.265)	(3.18)	(24°C)
77.46	2202	5.26	0.476	14.73	
(57.76)		(19.91)	(0.290)	(2.90)	Relative humidity
52.03	2233	4.28	0.576	12.17	47%
(38.81)		(16.18)	(0.351)	(2.40)	
26.27	2257	3.12	0.834	8.42	Barometer
(19.59)		(11.81)	(0.507)	(1.66)	28.86" Hg
1.03	2267	2.05	13.915	0.50	(97.73 kPa)
(0.77)		(7.77)	(8.464)	(0.01)	· /

Maximum Torque 407 lb/ft (552 Nm) at 1247 rpm

Maximum Torque Rise 40.3%

Torque rise at 1699 engine rpm 24%

	Dra	wbar P	erformanc	e Unbal	lasted —	Front Driv	ve Engag	jed	
					on Characte				
	Drawbar	Speed	Crank-		Fuel Consumption		Temp.	°F (°C)	Barom.
Power Hp	pull ls	mph	shaft		lb/hp.hr	Hp.hr/gal	cooling	Air dry	inch Hg
(kW)	(kN)	(km/h)	speed rpm	Slip %	(kg/kW.h)	(kW.h/l)	med	bulb	(kPa)
			Max	kimum Po	ower 11th G	ear			
96.26	6631	5.44	2099	2.24	0.490	14.33	190	76	28.87
(71.78)	(29.49)	(8.76)			(0.298)	(2.82)	(88)	(24)	(97.77)
			75% of Pul	l at Maxi	mum Power	· 11th Gear			
75.72	4967	5.72	2190	1.54	0.548	12.80	187	77	29.02
(56.47)	(22.09)	(9.20)			(0.333)	(2.52)	(86)	(26)	(98.27)
			50% of Pul	l at Maxi	mum Power	·11th Gear			
51.48	3313	5.83	2221	1.26	0.651	10.78	184	78	28.98
(38.39)	(14.73)	(9.38)			(0.396)	(2.12)	(84)	(26)	(98.14)
			75% of Pul	l at Redu	ced Engine	Speed 13th			
75.86	4950	5.75	1665	1.75	0.480	14.63	187	77	29.00
(56.57)	(22.02)	(9.25)			(0.292)	(2.88)	(86)	(25)	(98.21)
			50% of Pul	l at Redu	ced Engine	Speed 13th			
51.60	3310	5.85	1685	1.26	0.548	12.80	178	79	28.96
(38.48)	(14.72)	(9.41)			(0.333)	(2.52)	(81)	(26)	(98.07)

Table 2. Diesel fuel consumption	n for field operat	ions.					
		energy a					
	Range (gal/ac)						
Operation	Average	High	Low	Average from other states ^b			
Primary tillage							
Moldboard plow	1.81	3.50	0.90	1.87			
Chisel disk	1.36	3.50	0.80	1.09			
Offset disk	1.11	1.20	0.90	0.97			
Subsoiler	1.54	2.30	1.10	1.56			
Secondary tillage							
Disk	0.93	3.30	0.30	0.65			
Field cultivator	0.78	1.80	0.30	0.68			
Spring tooth harrow (drag)	0.73	1.80	0.20	0.48			
Fertilizer and chemical application							
Pesticide spraying	0.33	2.90	0.10	0.13			
Chemical incorporation	0.80	1.10	0.50	_			
Spreading fertilizer	0.30	0.50	0.10	0.19			
Knife in fertilizer	0.58	1.30	0.20	1.05			
Planting							
Row crop planter	0.51	1.00	0.20	0.54			
Grain drill	0.56	2.31	0.10	0.33			
Potato planter	0.95	1.90	0.90	0.95			
Broadcast seeder	0.28	1.12	0.10	0.15			
No-till planter	0.68	_	_	0.43			
Cultivation							
Cultivator	0.39	1.90	0.10	0.42			
Rotary hoe	0.23	0.70	0.10	0.21			
Forage harvesting							
Mower/conditioner	0.72	1.80	0.30	0.66			
Rake	0.46	1.26	0.20	0.24			
Baler	0.65	2.90	0.10	0.69			
Large round baler	0.80	_	_	_			
Forage harvester or green chop	1.57	2.00	0.20	1.87			
Corn silage harvester	3.14	6.70	1.70	2.69			
Grain and row crop harvest							
Small grain or bean combine	1.23	1.80	0.70	1.01			
Corn combine	1.51	2.20	0.70	1.37			
Corn picker	1.84	3.00	1.20	1.10			
Pull and window	0.52	1.10	0.30	0.34			
Beet harvester	1.37	1.90	0.90	1.91			
Topping beets	0.83	1.20	0.40	1.47			
Potato harvester	2.69	_	_	1.73			
PTO operated (gal/hr)							
Forage blower	2.19	6.20	0.90				
Irrigation	3.41	4.40	1.10	_			
Grinding	3.84	6.90	2.20	_			

^aHelsel, Z., and T. Oguntunde. 1985. Fuel requirements for field operations with energy saving tips. In *Farm Energy Use: Standards, Worksheets, Conserva-tion*, ed. C. Myers. East Lansing: Michigan State University. ^bIowa, Pennsylvania, Nebraska, Missouri, New York, Oklahoma, North Dakota, and Ontario, Canada

Appendix A. Determination of Coefficients in equation (4) for a "specific" tractor.

$$a = (Q_{75F} - Q_{50F})/[P_{PTO}(X_{75F} - X_{50F})]$$
(6)

$$b = (Q_{75F}/P_{PTO}) - aX_{75F}$$
(7)

$$c = (f/h - e/g)/(X_{75F} - X_{50F})$$
(8)

$$d = cX_{50F} + f/h \tag{9}$$

$$e = 1 - [(X_{75F} \cdot Q_{75R})/(X_{75R} \cdot Q_{75F})]$$
(10)

$$f = 1 - [(X_{50F} \cdot Q_{50R})/(X_{50R} \cdot Q_{50F})]$$
(11)

$$g = (RPM_{75F} - RPM_{75R})/RPM_{75F}$$
(12)

$$h = (RPM_{50F} - RPM_{50R})/RPM_{50F}$$
(13)

where

- Q = diesel fuel consumption (gal/h),
- X = ratio of equivalent PTO power to rated PTO power (decimal),
- N = ratio of reduced- and full-throttle engine speeds at operating load (decimal),
- *RPM* = engine speed for partial loads from full and reduced throttle (rpm),

 P_{PTO} = rated PTO power (hp).

The specific subscript refers to the condition maintained during the test. The tractor tests were conducted at full throttle (*F*) and reduced throttle (*R*) during the 50 percent and 75 percent drawbar load tests. For example, Q_{75F} is the fuel consumption at 75 percent drawbar load and full throttle. The calculations are based on fuel consumption data, power levels, and engine speed from four partial-load drawbar load tests and the PTO and drawbar power at rated engine speed. The equivalent power ratios (at the four test points) were computed from maximum drawbar power at rated engine speed and the current power level.

Below are the coefficients determined using equations (6) through (13) for the tractor test Summary 225 (Table 1):

P_{PTO}	= 115.96 hp
P_{PTO}	= 115.96 hp

$$P_{MaxDB} = 96.26 \text{ hp}$$

 $RPM_{50F} = 2,221 \text{ rpm}$ $Q_{50F} = 51.48 \text{ hp}/(10.78 \text{ hp-h/gal}) = 4.78 \text{ gal/h}$ $X_{50F} = 51.48 \text{ hp}/96.26 \text{ hp} = 0.535$

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RPM_{75F}	= 2,190 rpm
$Q_{75F} = 75.72 \text{ hp/(12.8 hp-h/gal)}$	= 5.92 gal/h
$X_{_{75F}} = 75.72 \text{ hp}/96.26 \text{ hp}$	= 0.787
RPM _{50R}	= 1,685 rpm

$$Q_{50R} = 51.60 \text{ hp}/(12.80 \text{ hp-h/gal}) = 4.03 \text{ gal/h}$$

$$X_{50R} = 51.60 \text{ hp}/96.26 \text{ hp} = 0.536$$

$$RPM_{75R} = 1,665 \text{ rpm}$$

$$Q_{75R} = 75.86 \text{ hp}/(14.63 \text{ hp-h/gal}) = 5.19 \text{ gal/h}$$

$$X_{75R} = 75.86 \text{ hp}/96.26 \text{ hp} = 0.788$$

$$a = \frac{5.92 \text{ gal/h} - 4.78 \text{ gal/h}}{115.96 \text{ hp} (0.787 - 0.535)} = 0.039 \text{ gal/hp-h}$$

$$b = \frac{5.92 \text{ gal/h}}{115.96 \text{ hp}} - (0.039 \cdot 0.787) = 0.0203 \text{ gal/hp-h}$$

$$e = 1 - \frac{0.787 \cdot 5.19 \text{ gal/h}}{0.788 - 0.92 \text{ gal/h}} = 0.1251$$

$$f = 1 - \frac{0.535 \cdot 4.03 \text{ gal/h}}{0.536 \cdot 4.78 \text{ gal/h}} = 0.1578$$

$$g = (2,190 - 1,665 \text{ rpm})/2,190 \text{ rpm} = 0.2397$$

$$h = (2,221 - 1,685 \text{ rpm})/2,221 \text{ rpm} = 0.2413$$

$$c = \frac{(0.1578/0.2413) - (0.1251/0.2397)}{(0.787 - 0.535)} = 0.5247$$

$$d = (0.5247 \cdot 0.5350) + (0.1578/0.2413) = 0.9345$$

Substituting the coefficients a, b, c, and d in equation (4), the final prediction equation for JD7610 becomes:

$$Q = (0.0390X + 0.0203) \cdot [1 - (N - 1) \cdot (0.5247X - 0.9345)] \cdot 115.96 \text{ hp}$$

This equation can be used to predict the fuel consumption when a JD7610 is used for different field operations with varying loads and engine speeds. An Excel spreadsheet (figure A-1) may also be used for computing coefficients using data from the Nebraska Tractor Test Laboratory report.

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1	Input		#		mary 225 for	OECD Tractor T John Deere 7610		INPUT
2	Rate Power			P _{pto} =			115.96	115.96 hp
3	Max Drawbar P	ower		P _{MaxDB} =			96.26	96.26 hp
4	Engine Speed @	50% Load, Full Throttle		RPM _{50F} =			2221	2221 rpm
5	Power @ 50% I	load, Full Throttle		$P_{50F} =$				51.48 hp
6	SFC @ 50% Lo	ad, Full Throttle		SFC _{50F} =			10.78	10.78 hp-hr/gal
7	Fuel Use @ 509	% Load, Full Throttle		$Q_{50F} = P_{50F} / SFC$	$C_{50F} = 51.48 \text{ hp} / 1$	0.78 hp-h/gal	4.78	4.78 gal/hr
8	Equivalent Pow	er @ 50% Load, Full Throttle		$\mathbf{X}_{50\mathrm{F}} = \mathbf{P}_{50\mathrm{F}} / \mathbf{P}_{\mathrm{Ma}}$	$_{\rm xDB} = 51.48 \text{ hp} / 96$	6.26 hp	0.535	0.535
9	Engine Speed @	75% Load, Full Throttle		$RPM_{75F} =$			2190	2190 rpm
10	Power @ 75% I	Load, Full Throttle		P _{75F} =			75.72	75.72 hp
11	SFC @ 75% Lo	ad, Full Throttle		SFC _{75F} =			12.8	12.8 hp-hr/gal
2	Fuel Use @ 759	% Load, Full Throttle		$Q_{75F} = P_{75F} / SF$	$C_{75F} = 75.72 \text{ hp} / 1$	2.8 hp-h/gal	5.92	5.916 gal/hr
13	Equivalent Pow	er @ 75% Load, Full Throttle		$X_{75F} = P_{75F} / P_{Ma}$	$_{\rm xDB} = 75.72 \text{ hp} / 96$	5.26 hp	0.787	0.787
14	Engine Speed @	50% Load, Reduced Throttle		RPM _{50R} =			1685	1685 rpm
15	Power @ 75% I	Load, Full Throttle		P _{50R} =			51.6	51.6 hp
6	SFC @ 75% Lo	ad, Full Throttle		SFC _{50R} =			12.8	12.8 hp-hr/gal
17	Fuel Use @ 509	% Load, Reduced Throttle		$Q_{50R} = P_{50R} / SP$	$C_{50R} = 51.60 \text{ hp} /$	12.80 hp-h/gal	4.03	4.031 gal/hr
8	Equivalent Pow	er @ 50% Load, Reduced Thrott	le	$X_{50R} = P_{50R} / P_{M}$	$_{\rm axDB} = 51.60 \text{ hp} / 9$	6.26 hp	0.536	0.536
9	Engine Speed @	75% Load, Reduced Throttle		RPM _{75R} =			1665	1665 rpm
20	Power @ 75% I	load, Full Throttle		P _{75R} =			75.86	75.86 hp
21	SFC @ 75% Lo	ad, Full Throttle		SFC _{75R} =			14.63	14.63 hp-hr/gal

Figure A-1. Spreadsheet for calculating the coefficients *a*, *b*, *c*, and *d* for equation (4).